

APPLICABILITY OF USING THERMAL STORAGE MATERIALS IN SOLAR AIR HEATER

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ABSTRACT

Solar air heater (SAH) is more suitable for residential space heating in winter season and in drying crops. This work presents thermal performance of flat plate SAH with and without thermal storages. A forced convection solar collector (FCSC) integrated with different sensible heat storage materials (sand, concrete, pure cement) has been developed and tested to evaluate its performance under meteorological conditions of Baghdad, Iraq. System consists of three flat plate solar air heaters with different heat storage materials, and the fourth without storage heat material. All collectors are supplied with centrifugal blower. The experimental results indicated that the pure cement gives better thermal performance than other storage heat materials (sand, concert), the pure cement gives heat along (240 min) after (6:30 PM), but the duration was (170 min) and (65 min) for concrete and sand respectively. The air flow rate is more affect on both outlet air temperature storage heat times, where studied the effect of three air velocities (0.8m/s, 1.4m/s and 2.1m/s) on SAH performance with best thermal storage heat material (pure cement), the increase in outlet air velocity leads to decrease in both outlet air temperature and storage time of thermal energy.

Keyword: Solar air heater, Thermal storage material, Flat plate solar collector.

امكانية استخدام المواد الخازنة للحرارة في السخان الشمسي الهوائي

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الخلاصة :

السخان الشمسي الهوائي مناسب للتدفئة السكنية في فصل الشتاء و كذلك في تجفيف المحاصيل. يبين هذا العمل الاداء الحراري للسخان الشمسي الهوائي مستوي الصفيحة مع المواد الخازنة للحرارة وبدونها. دمج مجمع الطاقة الشمسية ذو انتقال الحرارة بالحمل القسري مع مواد الخزن الحراري المختلفة (رمل، خرسانة، سمنت نقي) وقد تم تطوير وتقييم ادائها عند الظروف الجوية لمدينة بغداد،العراق. يتكون النظام من ثلاثة سخانات شمسية للهواء مستوية الصفيحة مع مواد الخزن الحراري المختلفة، والرابع بدون مادة خزن حراري. جميع المجمعات جهزت بمروحة طاردة مركزية. بينت النتائج العملية ان السمنت النقي اعطى افضل اداء حراري من بقية المواد الخازنة للحرارة (رمل، كونكريت)، اعطى السمنت النقي حرارة على طول (240 دقيقة) بعد (6:30 مساءً)، بينما كانت المدة (170 دقيقة) و (65 دقيقة) للكونكريت و الرمل على التعاقب. ان معدل جريان الهواء يؤثر على كل من درجة حرارة الهواء الخارج و اوقات الخزن الحراري، حيث تم دراسة تاثير ثلاث سرع للهواء الخارج (0.8 م/ثا، 1.4 م/ثا و 2.1 م/ثا) على اداء السخان الشمسي الهوائي مع افضل مادة خزن حراري (سمنت نقي)، ان زيادة سرعة الهواء الخارج من السخان الشمسي الهوائي يؤدي الى انخفاض كل من درجة حرارة الهواء الخارج و زمن التخزين للطاقة الحرارية.

الكلمات المفتاحية: السخان الشمسي الهوائي، مواد الخزن الحراري، المجمع الشمسي مستوي الصفيحة .

NOMENCLATURE

A_c = the area of collector (m^2).

C_p = the specific heat of air at constant pressure (1.008 kJ/kg.K).

\dot{m} = the mass flow rate (kg/s).

SAH = the solar air heater.

T_{in} = the inlet temperature of the cooling fluid ($^{\circ}C$).

T_{out} = the outlet temperature of cooling fluid ($^{\circ}C$).

\dot{V} = the volume of the air leaving the collector by the duct with diameter d (m^3/s).

v = the air speed in (m/s).

ρ = the density of air in (kg/m^3).

P = the air pressure (101325 Pa).

R = the specific gas constant for dry air (287.05 J/kg.K).

η_{th} = the solar air heater (SAH) efficiency.

Q_u = the useful heat gain.

1- INTRODUCTION

Solar air heater (SAH) is extensively used in industrial, residential and agricultural fields. Experimental studies reported various methods used for drying of agricultural materials using solar drier for pineapple drying by (Sodha, et al,1985), for onion drying by (Sarsavadia, 2007), and for copra drying by (Mohanraj & Chandrasekar, 2009). (Hassan Fath, 1995) studied the thermal performance of four common types of single pass SAH.

Common sensible heat storage materials used to store sensible heat such as water, gravel bed, sand, clay, concrete, etc, has been studied (Abqul-Enein, 2000). Forced convection solar driers seem to be an advantage over traditional methods and improve quality of product considerably by (Midilli, 2001). (Pangavhane & Sawhney, 2002) studied Heat storage materials for copra drying of a flat plate SAH.

On heat transfer characteristics of SAH, (Jain & Tiwari, 2003) studied effect of collector aspect ratio on that plate upward baffled SAHs. Normally thermal storage systems are employed to store heat, which includes sensible and latent heat storage by (Hawladar, 2003). The relationships between the direct solar irradiation, the heat flow resulted, the air velocity at the outlet, the air flow rate, the time of input in nominal regime of the collector and efficiency of conversion of solar energy into thermal energy are highlighted by (Sanda Budea, 2014).

This study presents development of a forced convection solar collector (FCSC) integrated with and without sensible heat storage material (pure cement, concrete and sand), to show the ability of using flat plate solar air heater to store heat energy to recharge it at night, and to show the affect of outlet air velocity on solar air heater performance with best storage material under meteorological conditions of Baghdad, Iraq at February 2015.

2-EXPERIMENTAL SECTION

2.1 Experimental Setup

In (Fig.1) shows the four flat plate air solar collectors. Each collector [area (0.6×1 m²)] with aluminum absorber plate (0.7 mm thick) black painted to absorb incident solar radiation. Absorber plate is placed directly behind transparent cover (glass) with a layer of air separating it from cover. Air to be heated passes between transparent cover (glass) and absorber plate. To increase temperature of air by green house effect, a glass cover (4 mm thick) is placed. Gap (100 mm) between glass and absorber surface is maintained for air circulation. Top side of collector is connected to (3) inch diameter centrifugal blower using reducer (Fig.2), and bottom side is opened with (3) inch diameter to make air circulation. Gap (40 mm) between absorber and back insulator for each collector, three collectors are filled with heat storage materials (sand, concrete, pure cement) to store heat energy at sunny hours and reuse it after sunset as shown in (Fig.3), and the 4th collector was without storage material, the thermo physical specification of heat storage materials are showed in (Table 1). The principle operation is shown in schematic diagram in (Fig.4). A side by side test of collectors is conducted to show the effect of the type of storage material on heat storage at the same environment condition (irradiation, ambient temperature, and wind speed). SAH is tilted to 65° with respect to horizontal. System is oriented to face south to maximize solar radiation incident on solar collector. The measurements were done in ALmashtal, Baghdad, Iraq (latitude 33.33158°N, longitude 33° E), the first test was done on 8 February to select the best storage heat material; the next test was done in 11 February to show the effect of outlet air velocity on solar air heater performance.

2.2 Experimental instruments measurement

Solar irradiation was measured with an (Protek / DM-301) instrument as shown in (Fig. 5). The response time of this instrument is 1s. The instrument measures solar intensity in the range (0 to 2000 Watt/m²).

Air velocity is measured by anemometer type (Kaindl /wind master) as shown in (Fig 2).

The absorber surface temperature, air inlet, and outlet temperature are measured by three k-type thermocouple with temperature data logger type (BTM-4208SD) as shown in (Fig.5). K-type thermocouples are suitable for measurements in the range at (-180 to +1300 °C).

2.2.1 Experimental Measurements Accuracy

- 1- The accuracy of solar irradiation meter is ($\pm 0.7\%$).
- 2- The accuracy of air velocity meter is ($\pm 4\%$).
- 3- The accuracy of temperature data logger is ($\pm 0.01\%$).
- 4- The accuracy of K-type thermocouples is ($\pm 0.4\%$).

3- MATHEMATICAL MODELING :-

Equation (1) was applied to estimate the solar heat gain delivered by the SAH:

$$Qu = \dot{m} * Cp * (T_{out} - T_{in}) \quad (1)$$

The air mass flow rate in (kg/s) is given by equation (2):

$$\dot{m} = \rho * V = \rho * \frac{\pi d^2}{4} * v \quad (2)$$

The efficiency of the SAH was calculated as equation (3):

$$\eta_{th} = \frac{Qu}{G * A_c} \quad (3)$$

The density of dry air can be calculated using the ideal gas law, expressed as a function of temperature and pressure as equation (4):

$$\rho = \frac{P}{R * T_{amb}} \quad (4)$$

4- DISCUSSION OF THE RESULTS :-

Fig.(6) shows the effect of using different storage materials (sand, concrete, pure cement) on the heat energy store. SAH without storage material give higher outlet temperature compared with others, this temperature is dropped fast when the irradiation is dropped after (12:30 PM) as shown in Fig.(7). But the others SAHs with different storage materials at the same day test gives lowest outlet air temperature depending on the type of storage material. Since the sum of the temperature goes to heated storage material from the absorber at sunny hours, and this leads to droop in absorber temperature depending on the ability of these materials to absorbing heat. But at sunset hours the temperature is transferred from the storage materials to absorber and led to heat outlet air. The results gives that the pure cement is the best for storage heat comparison with sand and concrete, the pure cement gives heat a long (240 min) after (6:30 PM), but concrete gives about (170 min), and sand (65 min) only.

Fig.(8) shows the heat gain for the four SAHs, the best results obtained are for pure cement storage material compared with others (sand, concert). Fig.(9) shows efficiency comparison among four SAHs depending on equation (3), before (2:30 PM) SAH without

storage materials gives highest efficiency, but SAH with pure cement storage material give lowest efficiency. SAHs efficiency with storage materials are raising with time (after 2:30 PM) because the storage materials reject the store heat energy to the absorber with dropping in absorbed irradiation, and the results shows that the SAHs with pure cement gives best behaviors with time comparison with sand and concrete. The above results was done on 8 February 2015

After detection from this experiment that the pure cement is best for storage heat compared with others, the next test was done on 11 February 2015 by using three solar air collectors with pure cement as storage heat material, The centrifugal blower of each collector was adjusted to give (0.8m/s, 1.4m/s, and 2.1m/s) respectively. Fig. 10 shows the effect of air velocity on outlet air temperature from the solar air collectors, the increase in air velocity led to decrease in both outlet air temperature storage heat time, the physical explain for this case are the increase in air velocity led to more heat transferred from absorber to air comparison with others air velocities, this increasing in air velocity is effect on heat gain for these collectors as shown in Fig. (11).

Fig.(12) shows the effect of outlet air velocity on instantaneous efficiency on 11 February 2015 at (12:30PM) with pure cement storage material; the (1.4m/s) gives best instantaneous efficiency comparison with others.

5- CONCLUSIONS :-

The different storage materials are affect on SAH behavior. The results indicated that when sand, concrete and pure cement are used in SAHs under the same environment condition, the outlet air temperature is less than the SAH without storage material, because of charging heat in the storage heat materials from the absorber at sunny hours, and the temperature is rejected back to absorber at sunset hours and used to heated air. The experimental results show that the pure cement gives the best behavior for storage heat along (4 hours) after (6:30 PM), but concrete about (2 hours and 50 minutes), and sand (1 hour and 5 minutes) only. The air flow is more effect on both storage heat and out let air temperature.

(Table 1) Thermo physical specification of heat storage materials

Material	Specific heat (J/kg.K)	Thermal conductivity (W/m.K)	Density (kg/m ³)
Sand-dry	800	0.32	1600
Cement-dry	1550	0.29	1506
Concrete	880	1.36	2307



Fig. (1) Experimental setup



Fig. (2) Air velocity meter with centrifugal blower



Fig. (3) Samples of storage heat materials used in this work

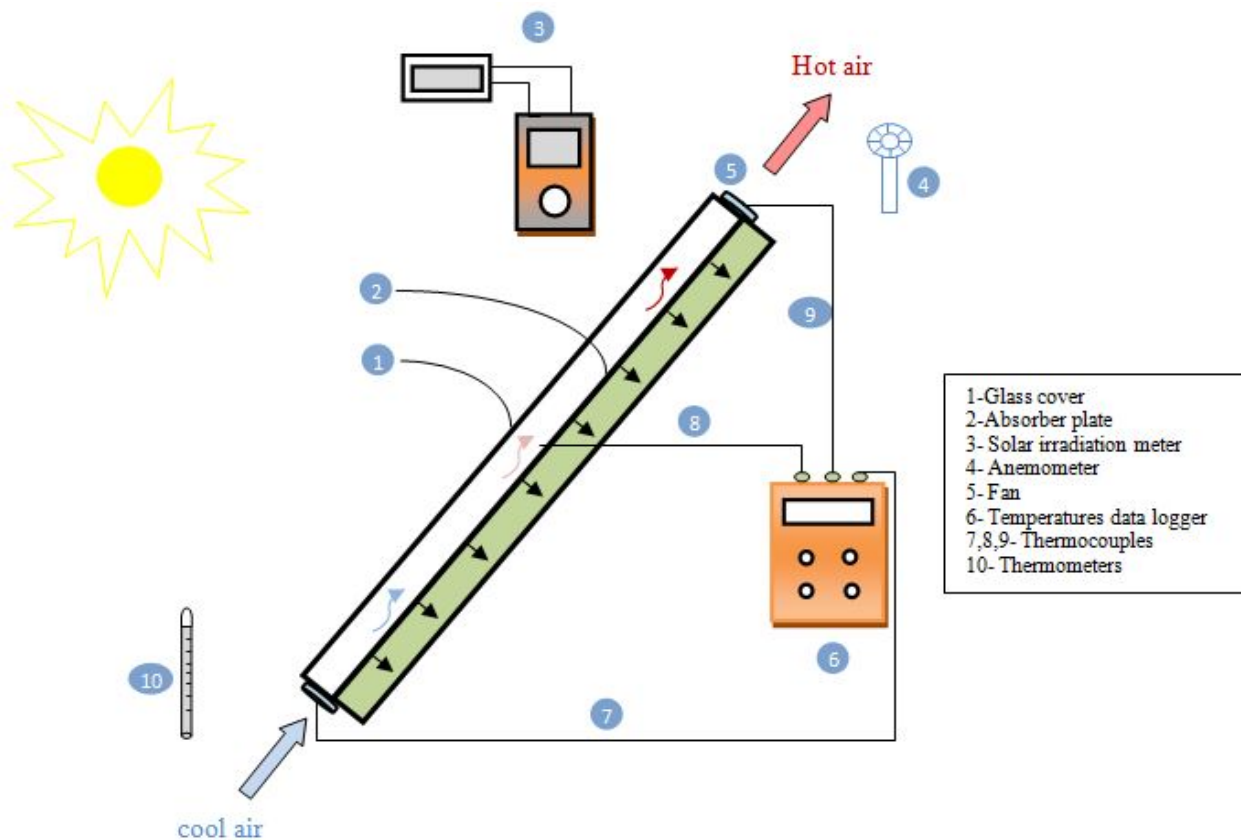


Fig. (4) Schematic diagram of the SAH principle operation

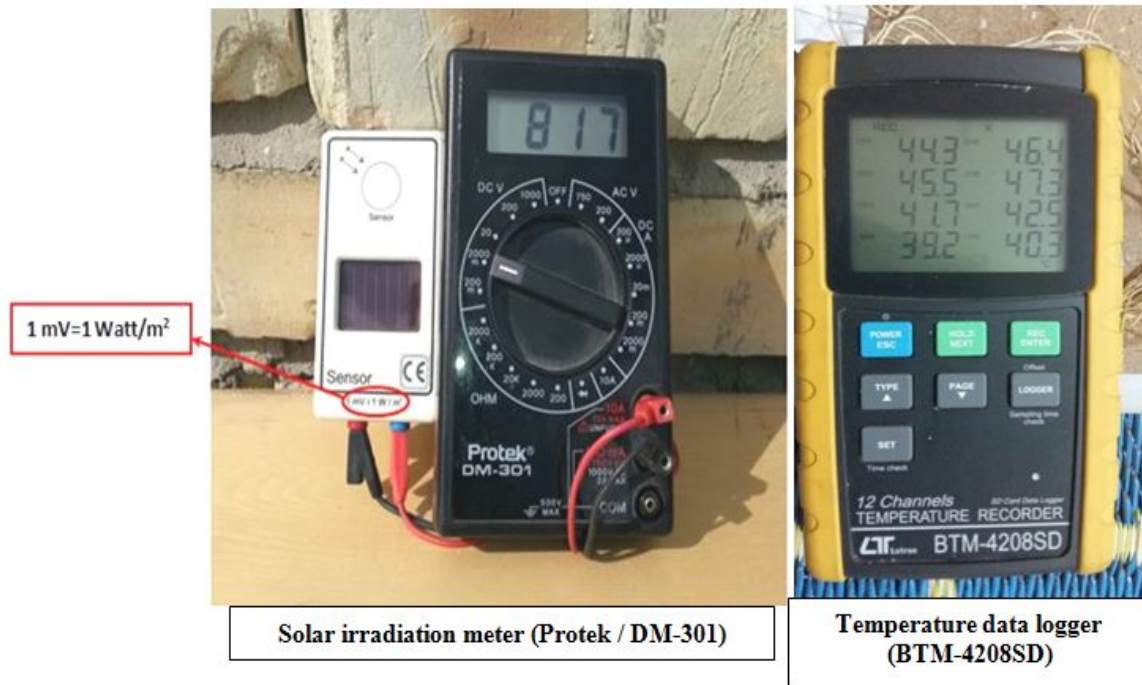


Fig. (5) Solar irradiation meter with temperature data logger

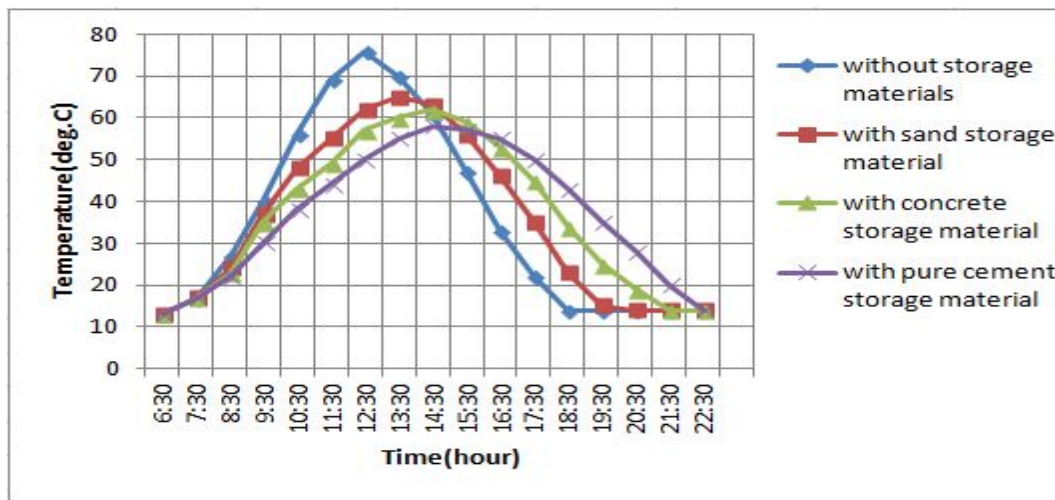


Fig. (6) Variation of outlet air temperature from SAHs on 8 February 2015 with air velocity 0.8 m/s



Fig. (7) Distribution of solar irradiation on 8 February 2015

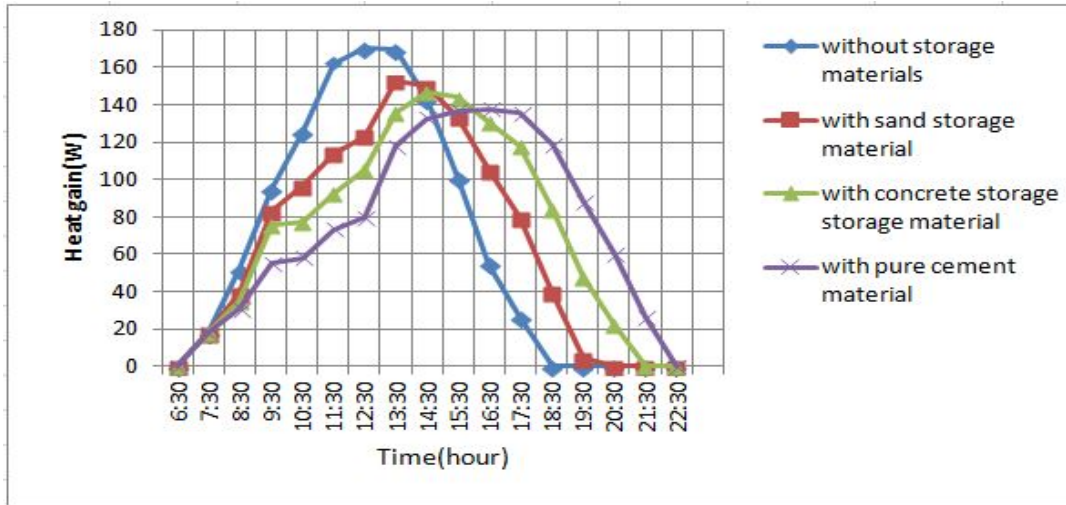


Fig. (8) Outlet SAHs heat gain on 8 February 2015 air velocity 0.8 m/s

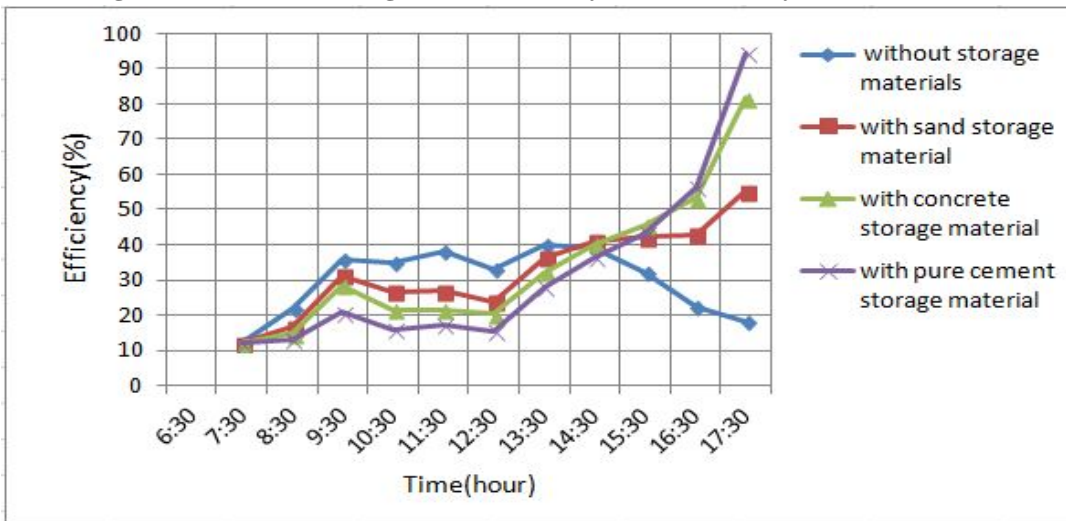


Fig. (9) SAHs efficiency on 8 February 2015 air velocity 0.8 m/s

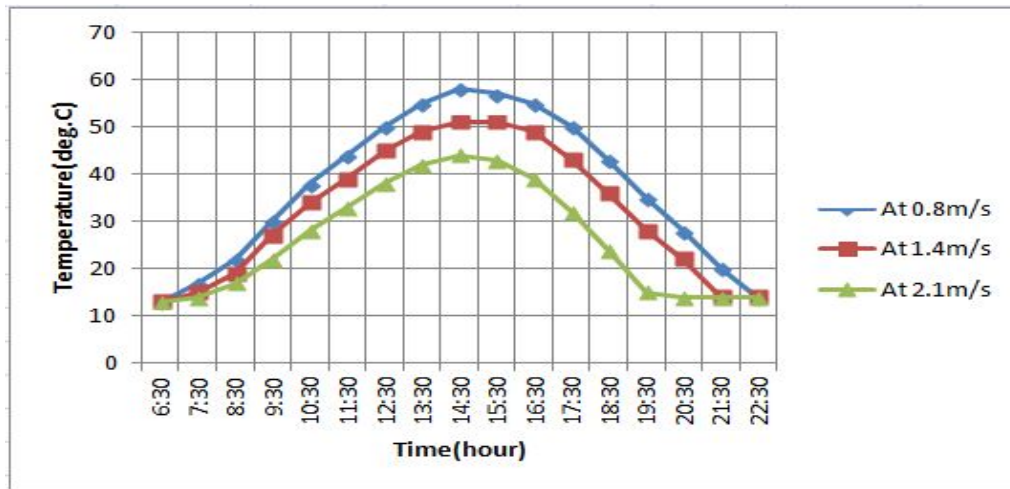


Fig. (10) Effect of air velocity on outlet air temperature with pure cement storage material on 11 February 2015

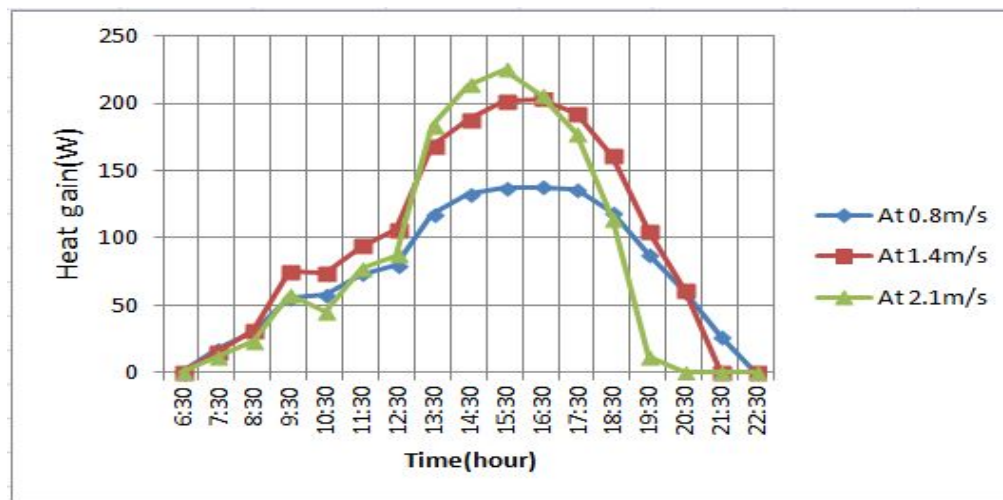


Fig. (11) Effect of outlet air velocity on heat flux with pure cement storage material on 11 February 2015

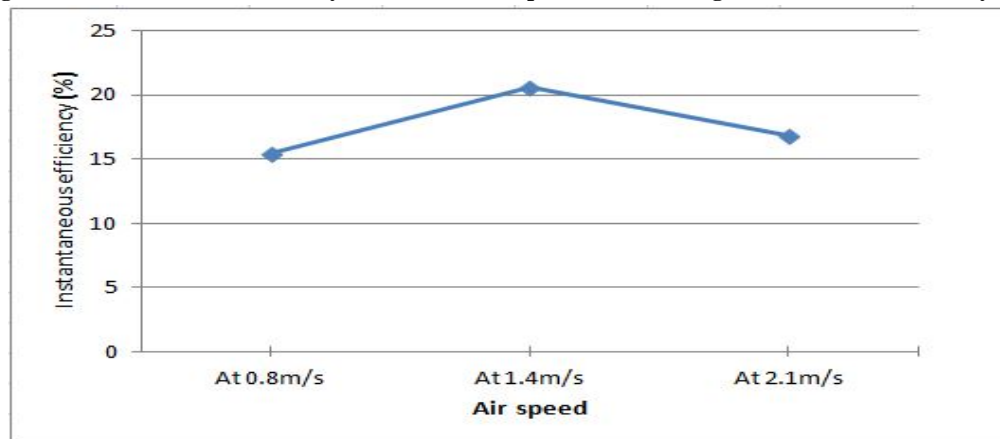


Fig. (12) Effect of outlet air velocity on instantaneous efficiency on 11 February 2015 (12:30PM) with pure cement storage material

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